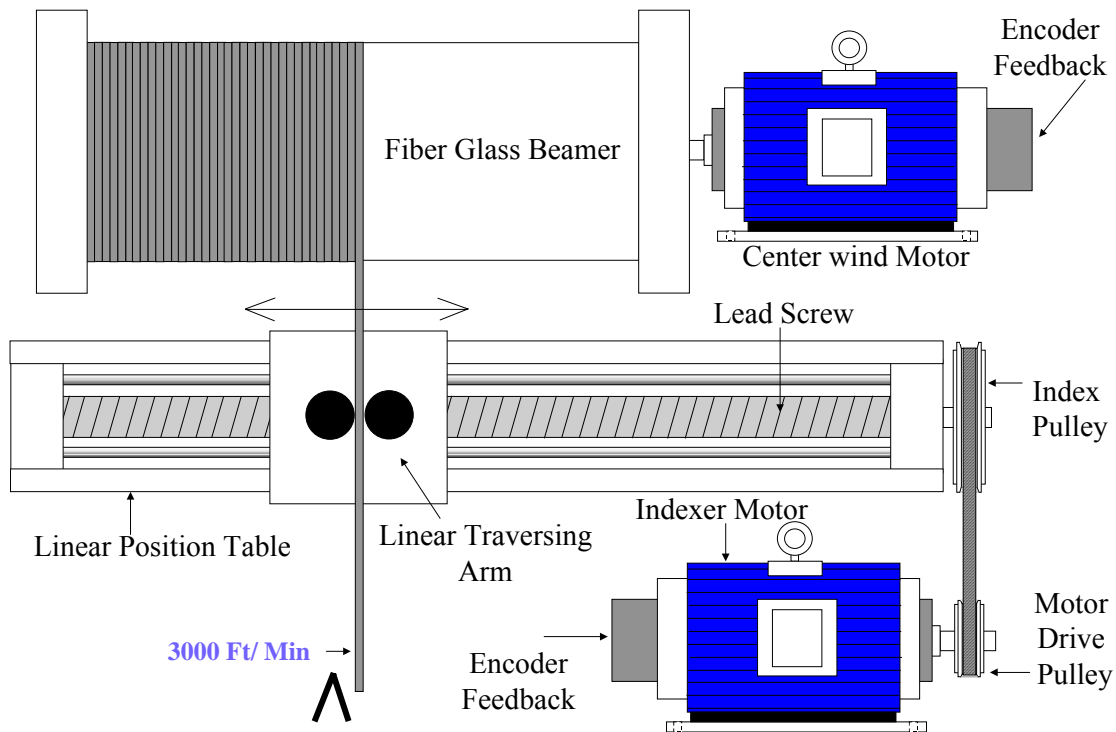


This form is used to help size common indexing applications. The form contains: Indexing overview, sizing example, and customer application data.



Indexing is a method of providing position and velocity control of a machine section or workpiece. Traditionally, Indexer control had been accomplished using stepper and servo controls. With the improvements of Flux Vector AC drives, which regulates the magnetic flux and torque generating current of induction motor, position and velocity control can be accomplished using the less costly Flux Vector drive.

The Servo or Vector drive controls the indexing motion by controlling the motor to move a predefined distance and velocity. An optical encoder or resolver is used to close the position and velocity loop during a move profile. The processor move command will be broken down into a specific number of pulses. During the move command, the encoder pulses provide feedback. By using **CASE** application software, the AC vector drive can count the number of pulses.

The predefined profiles are stored in the software, and the following parameters can be programmed.

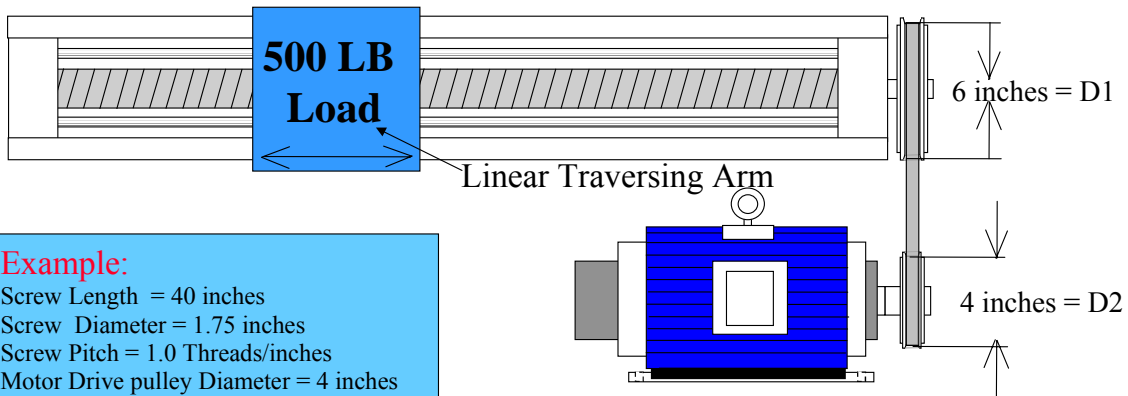
<u>Parameters</u>	<u>Communication Control</u>
Speed1, Home speed	Modbus
Preset Speed 1,2,3	Modbus-Plus
Acceleration	RS232-RS485
Deceleration	Profibus
Distance	Interbus
Dwell	
Home sequence	

Motion System Performance Comparison

<u>Type</u>	<u>V/f Control</u>	<u>Vector Control</u>	<u>AC Servo</u>
HP Range	0.5-1500 HP	0.5-1500HP	1-100 HP
Speed Range control	40:1	1000:1	200,000:1
Speed control accuracy	± 2%	± .01%	± .01%
Frequency control	1- 400 Hz	1- 400 Hz	1- 400 Hz
Torque Range		.01-150%	.01-200%
Torque Control	No	Yes	Yes
Velocity control	No	Yes	Yes
Positioning	No	Yes	Yes
Repeatability	Not Possible		± 1 arcmin
Motor Inertia:Load Intertia	N/A		10:1-200:1

The application below requires calculations that are found on the next page. There are many different sizing programs in the market, but without having the basic understanding of the calculations, a critical profile or mechanical question could be missed.

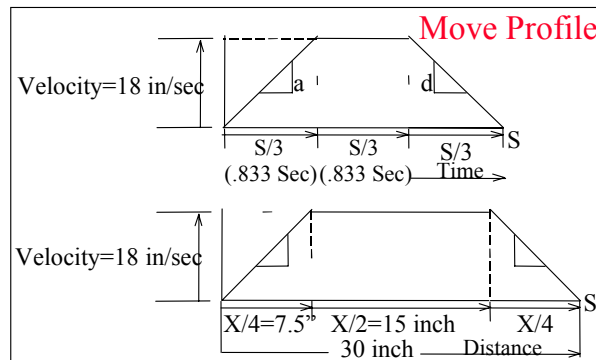
The calculations require a few assumptions. The velocity & distance profiles are broken up into 3 sections. If you make the assumption that distance is $x/4$ (x is distance) for acceleration & deceleration, and $x/2$ for constant velocity, the Time profiles can be broken into $s/3$ sections (s is time). The profiles are given in the **Move Profile** below.



Example:

Screw Length = 40 inches
 Screw Diameter = 1.75 inches
 Screw Pitch = 1.0 Threads/inches
 Motor Drive pulley Diameter = 4 inches
 Index table pulley Diameter = 6 inches
 W1 = Weight Load pulley=4 lb
 W2 =Weight Motor Pulley = 9lb
 Motion requirement = 30 Inches
 Speed requirement = 2.5 seconds;
 Distance X/4= a=-d & Distance X/2 Run

- a = -d = Acceleration = _____ in/sec²
- v = Velocity = _____ in/sec
- p = Screw pitch _____ Threads/Inch
- DS = Screw Diameter _____ inches
- W = Load _____ LBS
- F = Breakaway Force _____ LBS
- e = Efficiency of screw _____ %
- L=Length of Screw _____ Inches
- J = inertia _____ oz-in²
- D1 = Dia Load pulley _____ inches
- D2 = Dia Motor Pulley _____ inches
- W1 = Weight Load pulley 4 lb
- W2 =Weight Motor Pulley 9 lb



Formula

Sizing Example

$$\text{Acceleration} = a = -d = \frac{2X}{t^2} = \frac{2(X/4)}{(S/3)^2} = \frac{4.5X}{S^2}$$

Steps ① $a = -d = \frac{4.5X}{S^2} = \frac{4.5 * 30}{2.5^2} = 21.6 \text{ in/sec}^2$

$$\text{Velocity} = at = \frac{4.5X}{S^2} * \frac{S}{3} = \frac{1.5X}{S}$$

② $v = at = 1.5 (30 \text{ inch})/2.5 \text{ Sec} = 18 \text{ inch / Sec}$
 $18 \text{ in/sec} * 6 \text{ in} * 60 \text{ sec}$

$$\text{Max RPM}_{\text{motor}} = \frac{\text{Max Speed} * \text{Lead Screw Pulley Dia} * (60 \text{ sec})}{\text{Pitch Dia} * \text{Motor Pulley Dia} * (\text{min})}$$

③ $\text{RPM}_{\text{motor}} = \frac{18 \text{ in/sec} * 6 \text{ in} * 60 \text{ sec}}{1 \text{ in/rev} * 4 \text{ in} * \text{min}} = 1620 \text{ RPM}$

$$\text{Torque}_{\text{total}} = (\text{Torque}_{\text{Friction}} + \text{Torque}_{\text{Acceleration oz - in}}) 1.10$$

④ $F = \text{Force}_{\text{Friction}} = .25 * 500 \text{ lb} * 16 \text{ oz/lb} = 2000 \text{ oz}$

$$\text{Torque}_{\text{Friction}} = \frac{F(D2/D1)}{2\pi * p * e} \quad \& \quad F = \text{Force}_{\text{Friction}} = u_s * W$$

⑤ $\text{Torque}_{\text{Friction}} = \frac{(4/6) 2000 \text{ oz}}{2\pi * 1.0 \text{ threads/in} * .65} = 326 \text{ oz-in}$

$$J_{\text{Load @ motor}} = \frac{W(D2/D1)^2}{(2\pi p)^2}$$

⑥ $J_{\text{Load @ m}} = \frac{500 \text{ lb} * 16 \text{ oz/lb} * (4/6)^2}{(2\pi * 1.0 \text{ threads/in})^2} = 90.15 \text{ oz-in}^2$

$$J_{\text{lead screw @ motor}} = \frac{\pi L \rho R^4 (D2/D1)^2}{2} \quad ; \quad \rho = \text{Steel density} = 4.48 \text{ oz/in}^3$$

⑦ $J_{\text{lead screw @ m}} = \frac{\pi * 40 \text{ in} * 4.48 \text{ oz/in}^3 * (1.75 \text{ in}/2)^4 * (4/6)^2}{2}$

$$= 73.29 \text{ oz - in}^2$$

Customer Data

$$J_{\text{load pulley @ motor}} = \frac{W_{\text{load Pulley}} R_{\text{load Pulley}}^2 * (D2/D1)^2}{2}$$

⑧ $J_{\text{load p @ motor}} = \frac{9 \text{ lb} * 16 \text{ oz/lb} * 3^2 * (4/6)^2}{2} = 288 \text{ oz-in}^2$

$$J_{\text{Motor Pulley @ motor}} = \frac{W_{\text{Motor Pulley}} * R_{\text{load Pulley}}^2}{2}$$

⑨ $J_{\text{Motor Pulley @ motor}} = \frac{6 \text{ lb} * 16 \text{ oz/lb} * 3^2}{2} = 432 \text{ oz-in}^2$

$$\omega_{\text{@ motor}} = 2\pi * p * \text{velocity}_{\text{@ t=.833 sec}} * (D1/D2) = \text{angular velocity}$$

⑩ $\omega_{\text{@ motor}} = 2\pi * 1.0 \text{ thread/in} * 18 \text{ in/sec} * (6/4)$

$$J_{\text{Motor @ 1.5hp given in Marathon catalog}} = 0.14 \text{ lb-ft}^2$$

⑪ $J_{\text{Motor}} = 0.14 \text{ lb-ft}^2 * 16.0 \text{ oz/lb} * (12 \text{ in}/1 \text{ ft})^2 = 322.56 \text{ oz-in}^2$
 $= 169.56 \text{ radians/sec}$

$$T_{\text{Accel}} = \frac{\omega (J_{\text{Load}}/e + J_{\text{lead screw}} + J_{\text{load pulley}} + J_{\text{Motor Pulley}} + J_{\text{Motor}})}{gt}$$

⑫ $T_{\text{Accel}} = \frac{169.56 (90.15 / .65 + 73.29 + 288 + 432 + 322.56)}{386 \text{ in/sec}^2 * .833} = 661 \text{ oz-in}$

$$\text{Torque}_{\text{total}} = (\text{Torque}_{\text{Friction}} + \text{Torque}_{\text{Acceleration oz - in}}) 1.10$$

⑬ $\text{Torque}_{\text{total}} = (326 \text{ oz-in} + 661 \text{ oz-in}) 1.10 = 1085 \text{ oz-in}$

$$\text{HP} = \frac{\text{Torque}_{\text{total}} * (\text{RPM max})}{5252 * 12 \text{ in/lb} * 12 \text{ in/ft}}$$

⑭ $\text{HP} = \frac{1085 \text{ oz-in} * (1620 \text{ RPM})}{5252 * 16 \text{ oz/lb} * 12 \text{ in/ft}} = 1.75 \text{ HP}$

Company Name	<input type="checkbox"/> End user <input type="checkbox"/> Distributor <input type="checkbox"/> OEM
Contact Name #1	Contact Name #1 e-mail
Contact Name #2	Contact Name #2 e-mail
Address	City
State	Zip
Phone	Fax

Machine Data

Type of application (Lead Screw, Conveyer, Rack and pinion, Turntable, Machine tool) _____

Material moved _____

p = Screw pitch _____ Threads/Inch

DS = Screw Diameter _____ inches

W = Load _____ LBS

F = Breakaway Force _____ LBS

e = Efficiency of screw _____ %

L=Length of Screw _____ Inches

J = inertia _____ oz-in²

Transmission Data, Pulley or chain and sprocket

D1 = Diameter Load pulley or sprocket _____ inches

D2 = Diameter Motor Pulley or sprocket _____ inches

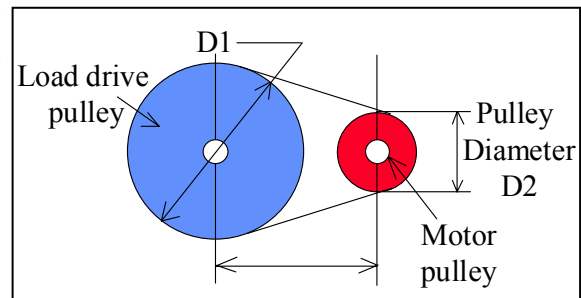
W1 = Weight Load pulley or sprocket _____ lb

W2 =Weight Motor Pulley or sprocket _____ lb

Inertia of Pulley or sprocket #1[^] _____ oz-in²

Inertia of Pulley or sprocket #2[^] _____ oz-in²

Efficiency _____ %



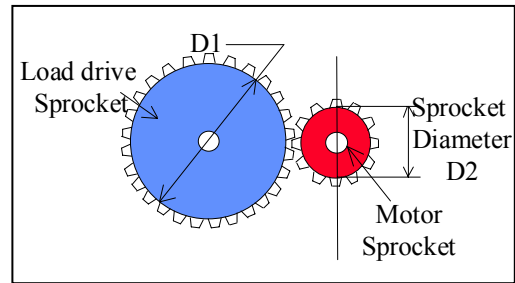
[^] If not known, then review example above

Gear Box

Gear Box ratio _____ Motor: Load
 Inertia of Gear Box^B _____ oz-in²
 Efficiency _____ %

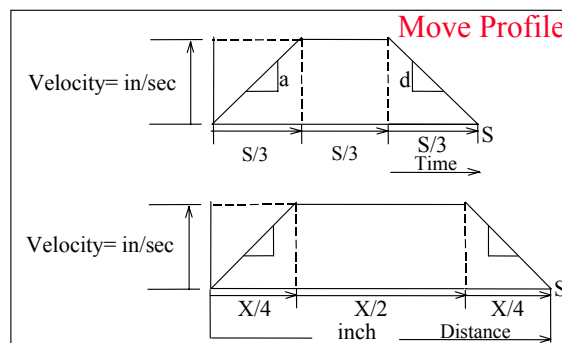
Gears

D1 = Diameter Load Gear _____ inches
 D2 = Diameter Motor Gear _____ inches
 Number of teeth Load Gear _____ teeth
 Number of teeth Motor Gear _____ teeth
 W1 = Weight Load pulley Gear _____ lb
 W2 =Weight Motor Gear _____ lb
 Inertia of Gear^A _____ oz-in²
 Inertia of Gear^A _____ oz-in²
 Efficiency _____ %



Move Profile

a = -d = Acceleration = _____ in/sec² If Acceleration = Deceleration (90% of Applications)
 Acceleration = _____ in/sec² If Acceleration ≠ Deceleration
 Deceleration = _____ in/sec² If Acceleration ≠ Deceleration
 Machine Design speed _____ (Inches/ Seconds)
 Minimum move profile _____ seconds



^A If not known, then review example above

Existing Power Distribution

- Isolation Transformer _____ KVA Primary Voltage _____ AC Secondary voltage _____ AC
- Line Reactors Impedence _____ (%) Load Reactors Impedence _____ (%)
- Dynamic Braking Resistor: Duty Cycle i.e. 3%, 5% _____ % Resistance _____ Ohms
- Dynamic Resistor Power rating _____ Watts

Drive Communication Requirements

- Modbus Plus Modbus Device Net Profibus Arcnet LAN Other _____

Drive Input Requirements

- Start Stop Forward Reverse Run
- Jog Preset Speed1 Preset Speed 2
- Other _____

Drive Output Requirements

- Drive alarm fault Drive severe fault Run Zero speed
- At speed Encoder feedback pass through (PGX card)
- Other _____

Analog Input

- speed reference 0-10VDC 4-20ma Other _____

Analog Output

- Drive Speed (Ft/minute) Bus Voltage Other _____

Special Types of Control

- Drive system start Drive system stop Regenerative to fast stop - full current limit or ramped
- DC Bus Over Voltage Suppression (Used to prevent overvoltage tripping from an unbalanced load)
- Other _____